



Electrostatic bellow muscle actuators and energy harvesters that stack up / Sirbu I.D.; Moretti G.; Bortolotti G.; Bolignari M.; Dire S.; Fambri L.; Veretny R.; Fontana M. *Journal of Intelligent and Robotic Systems*. - ISSN 2470-9476. - *ELETTRONICO*, 6:51(2021), pp. eaaz5796.1-eaaz5796.11. [10.1126/SCIROBOTICS.AAZ5796]

ARCHIVIO ISTITUZIONALE
DELLA RICERCA

Alma Mater Studiorum Università di Bologna Archivio istituzionale della ricerca

Electrostatic bellow muscle actuators and energy harvesters that stack up

This is the final peer-reviewed author's accepted manuscript (postprint) of the following publication:

Published Version:

Availability:

This version is available at: <https://hdl.handle.net/11585/870006> since: 2022-02-26

Published:

DOI: <http://doi.org/10.1126/SCIROBOTICS.AAZ5796>

Terms of use:

Some rights reserved. The terms and conditions for the reuse of this version of the manuscript are specified in the publishing policy. For all terms of use and more information see the publisher's website.

This item was downloaded from IRIS Università di Bologna (<https://cris.unibo.it/>).
When citing, please refer to the published version.

(Article begins on next page)

Electrostatic bellow muscle actuators and energy harvesters that stack up

I.D. Sîrbu¹, G. Moretti², G. Bortolotti¹, M. Bolignari¹, S. Diré¹, L. Fambri¹, R. Vertechy³, M. Fontana^{1,2†}.

† Corresponding Author: marco.fontana@santannapisa.it

¹ Department of Industrial Engineering, University of Trento, Trento, Italy

² TeCIP Institute, Scuola Superiore Sant'Anna, Pisa, Italy

³ Department of Industrial Engineering, University of Bologna, Bologna, Italy

Abstract

Future robotic systems will be pervasive technologies operating autonomously in unknown spaces that are shared with humans. Such complex interactions make it compulsory for them to be lightweight, soft, and efficient in a way to guarantee safety, robustness and long-term operation. Such a set of qualities can be achieved using soft multipurpose systems that combine, integrate and commute between conventional electromechanical and fluidic drives, as well as harvest energy during inactive actuation phases for increased energy efficiency.

Here, we present an electrostatic actuator made of thin films and liquid dielectrics, combined with rigid polymeric stiffening elements to form a circular electrostatic bellow muscle (EBM) unit capable of out-of-plane contraction. These units are easy to manufacture and can be arranged in arrays and stacks, that can be employed as a contractile artificial muscle, as pump for fluid-driven soft robots, and as energy harvester. As an artificial muscle, EBMs of 20-40 mm in diameter are able to exert forces of up to 6N, to lift loads over a hundred times of their own weight, to reach contractions of over 40% with strain rates over $1200\%s^{-1}$, with a bandwidth over 10Hz. As a pump driver, these EBMs produce flow rates of up 0.63 liters per minute and maximum pressure head of 6 kPa, while as generator, they reach a conversion efficiency close to 20%.

The compact shape, low cost, simple assembling procedure, high reliability and large contractions make the EBM a promising technology for high-performance robotic systems.

Summary

An electrostatic transducer based on dielectric polymers and fluids, capable to work as contractile actuator, pump and energy harvester

Introduction

Future robotic and mechatronic systems are becoming pervasive technologies and are foreseen to share spaces with humans and other living creatures. This has encouraged the pursuit of robot designs that are mechanically soft or compliant, able to adapt to and gently interact with the external world in the same fashion as living beings [1]. A key component of these robots is the actuation system, which is required to contribute to different motoric tasks, including soft joints motion/stretching, similar to mammalian muscles, or interaction with objects through inflatable/bendable interfaces [2], [3].

In response to this need, the last decade has seen a huge effort to develop fluidic actuation systems (FASs) [4], [5] and linear artificial muscles (AMs) [6], [7], [8] for soft robots. Robotic FASs are based on soft actuation structures capable of smooth interactions with the environment, but they require bulky compressors or pumps. Several works have been conducted to develop battery-based micro-compressors [4] and lightweight fully-polymeric pumps based on smart materials [5], [9]. The first have been successfully demonstrated in some robotic applications, but have limited efficiency, scarce controllability and they are inherently stiff and noisy. The latter rely on innovative operating principles such as electro-osmosis [10], thermo-pneumatic force [11] and charge-injection electrodynamics [5] and are suitable for very small scale applications with flow rates below a few ml/min, but they struggle to reach intermediate scales required for centimeter-scale robots.

In parallel, the last decade has also seen a huge effort to develop AMs able to produce displacements and forces in the range 10^{-3} - 10^{-1} m and 10^{-3} - 10^2 N respectively. Traditional multipole electrical motors struggle to reach these scales due to the complexity in miniaturization, whereas micro actuation systems [12] cannot be easily scaled-up. Although thermally-driven actuators [13], [14] have been identified as promising options, their response is limited by slow heat transfer dynamics.

Electrostatic actuators are an interesting option for actuation of robots thanks to their simple design (with few components), lightweight, high efficiency, ease in control, self-sensing capabilities and bidirectional operation (i.e. they can work as actuator, and as generator in the passive phase of motion). Electrostatic actuators that take advantage of Maxwell pressures within an air gap are the most effective solutions in the length-scale 1-10 μm (i.e. for micro electromechanical systems), but do not effectively scale-up in the mm-cm range due to the

decrease of the dielectric strength in large air-gaps. Dielectric elastomers (DEs) [15], [16], [17] have been introduced as a possible solution for electrostatic actuators upscaling. However, they require complex layouts with pre-stretched functional membranes [18] and their useful force is limited by the polymer mechanical stiffness.

In 2009, Prof. Najafi introduced the electrostatic hydraulic micro-actuator [19], [20], which relies on a combination of deformable dielectrics and a dielectric liquid gap. Compared to microactuators with air gaps, these systems can bear larger electric fields. Compared to DEs, they present little or no force limitation due to the elastic stresses. The full potential of this concept was later proven at larger scales by Prof. Keplinger with the development of the HASELs [21]. In parallel, Prof. Shea [22] introduced the idea of zipping electrostatic actuators, i.e. a device in which two electrode-coated dielectric polymeric films come into contact through a progressive increases of their contact surface, in response to an applied electric field. However, they obtained non-optimal performance due to the use of air as the dielectric fluid. Later on, taking advantage of dielectric liquids, Prof. Fontana [23] demonstrated the possibility to revert the zipping actuation principle to perform electrical energy generation.

Among the most recent actuator concepts based on the zipping kinematics and a liquid dielectric, Peano-HASEL [21], [24] and Electro-Ribbon AMs [25] demonstrated impressive results in terms of energy densities, on the order of 10^2 J/kg, despite the use of simple off-the-shelf low-cost materials such as polypropylene or other flexible polymeric films and dielectric oils. However, these devices were only demonstrated as AM actuators, and no proof of bidirectional actuator-generator operation was delivered.

Here, we present the electrostatic bellow muscle (EBM), a lightweight electrostatic transducer that combines fluid dielectrics, flexible electroactive films and stiffening elements to form a deformable unit which contracts and pumps the dielectric fluid in response to an electrical stimulus. Unlike previously proposed concepts, an EBM device can alternatively work as: (i) a hydraulically coupled actuator (which exploits the fluid pressure to make work on the external world), in the same fashion as HASELs [21]; (ii) a purely electrostatic actuator (without pressurization of the dielectric liquid), in a similar fashion as electrostatic ribbons [25].

As contractile muscles, EBMs offer (i) suitability to be combined in arrays to achieve large strokes and forces; (ii) contractions over 40%, comparable to those of natural muscles and exceeding those of several AM technologies [8], [26], [27]. As hydrostatic pump for FASs,

EBMs overperform state-of-the-art soft pumps based on dielectric polymers [28]. Compared to Peano-HASELs, which excel in strain rate and power density but require external rolling/translating mechanisms (such as pulleys) to achieve actuation strains over 20% [27], EBMs provide larger contractions and better ease of stacking. Compared to electrostatic ribbons, which are capable of huge contractions of up to 100% [25], EBMs reach larger strain rates and provide more stable encapsulation of the dielectric fluid. Additionally, EBMs contractile muscles can operate in generation mode, converting pulsating mechanical energy into electricity, without modifications to their layout and loading mode, reaching an efficiency of approximately 20%. Therefore, they could be effectively employed to implement power recycling during the passive/breaking phases of actuation.

As it stands, the EBM offers an extremely interesting perspective to enable direct-drive actuation of soft lightweight robots that combine the AMs and FASs. Switching from actuator to generator mode might offer further opportunities to improve energy efficiency and produce a drastic reduction of the required batteries capacity in autonomous robots.

Results

Working principle

The EBM is an electrostatic transducer that converts electrical energy into mechanical work (or vice-versa) taking advantage of electrostatic forces between oppositely charged electrodes [29], and it bears the ability to work both as an actuator and a generator [16], [24], [30], [23].

The EBM consists of a circular pouch made of two overlapped layers of compliant polymeric dielectric material holding opposite flexible electrodes on their external faces (Fig. 1A-D). The dielectric layers are held together by annular frames attached along their perimeter. These frames have the role of maintaining the structural integrity of the EBM and constraining its deformation. The pouch has a single central opening on the top and contains a volume of dielectric liquid. The bottom of the pouch holds a rigid end effector through for force application (Fig. 1B).

In the free state, the electrodes are flat and the polymeric dielectric layers are in contact. Once a load is applied, the membranes deform out-of-plane, originating a liquid-filled pouch with a shape similar to a double truncated cone shell.

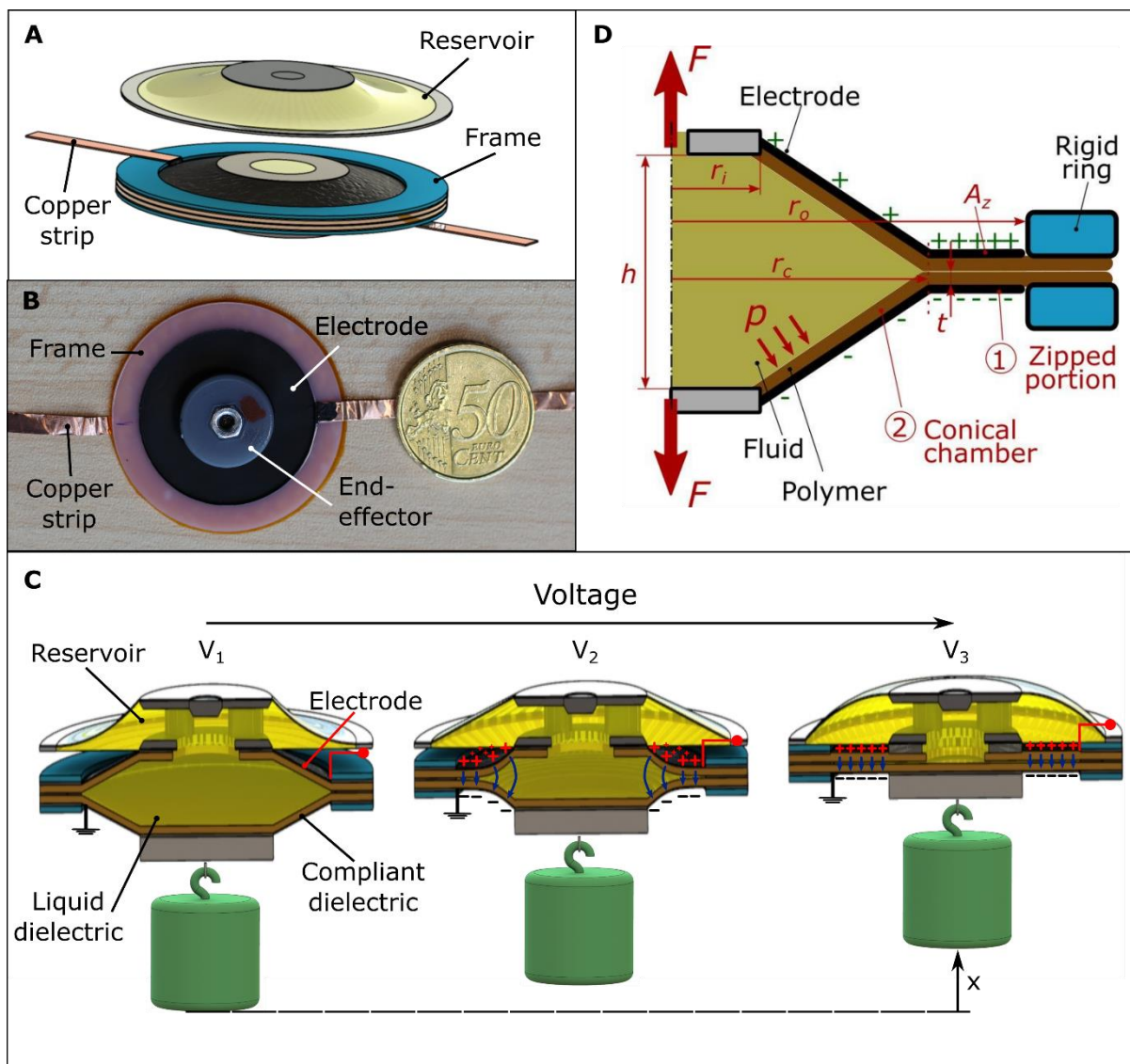


Fig. 1. Design and working principle of the EBM. (A) Representation of the EBM. (B) Photograph of an EBM prototype (bottom view) with reinforcing rings. (C) Schematic of the EBM with integrated fluidic reservoir, and phases of the actuation at three different voltages ($V_1 < V_2 < V_3$). In the initial state, the pouch is completely open under the action of a load. At the intermediate voltage, zipping takes place and the load is pulled up. At the maximum voltage, the device is fully zipped, and the liquid dielectric between the polymeric layers is minimum. Blue arrows in the picture qualitatively represent electric field streamlines. (D) Modelling of an EBM as a combination of two portions: a flat zipped portion, and a conical fluid chamber.

In the actuation mode (Fig. 1C), with the application of a voltage, charges on opposite electrodes attract, causing the polymeric layers to be pulled towards each other in a “zipping” motion [22]. During zipping, the polymeric layers gradually come into contact (starting from the perimeter) and liquid is evacuated through a central opening towards a reservoir.

Due to its layout, the EBM can pursue two different actuation modes: as a contractile actuator which makes work against an external force, or as a hydrodynamic actuator which

makes work against the pressure of the fluid. In the first mode, Maxwell stresses [16] are exploited to make work against a force applied at the end effector, whereas the dielectric fluid is evacuated towards a soft-walled variable volume reservoir, hence experimenting little or no variation in pressure with respect to atmospheric conditions. In the second mode, Maxwell stress is used to induce a significant pressurization in the fluid chamber and the fluid is delivered to a fluid utility.

In the generation mode, similar to DE generators [31], external mechanical forces drive the pouch deformation, while the voltage is properly varied in a such a way that electrical energy is generated and supplied to the driving electronics.

Modelling

The EBM can be modelled as a variable capacitor, using an approach similar to that presented in [29], introducing the following simplifications:

- In a generic configuration the EBM geometry can be ideally divided into two portions (Fig. 1D): a fully-zipped outer region, in which two flat dielectric polymer portions face each other over an annular surface; and a double-cone chamber in which the polymer layers enclose a volume of dielectric fluid. Further assumptions on the distribution of the strains in the dielectric polymer layers are discussed in the Supplementary Materials (SM).
- The electrical response is purely electrostatic, i.e., leakage currents through the dielectric layers are neglected. Similar to [29], the total capacitance of the device is approximately equal to that the zipped annular portion, whereas the capacitance of the double-cone portion housing the oil is negligible.

The EBM kinematics is described through two independent parameters, e.g., the height h of the pouch, and the radial distance r_c from the cone axis of the inner perimeter of the zipped region (Fig. 1D). The EBM response is governed by the following equations (see derivation and validation in the SM):

$$\left\{ \begin{array}{l} F + p \cdot \frac{\partial \Omega_c}{\partial h} = \frac{\partial U_{el}}{\partial h} + \epsilon_p E^2 \cdot A_z \frac{\partial t}{\partial h} \\ p \cdot \frac{\partial \Omega_c}{\partial r_c} = \frac{\partial U_{el}}{\partial r_c} + \epsilon_p E^2 \cdot \left[2\pi t r_c + A_z \frac{\partial t}{\partial r_c} \right] \end{array} \right. \quad (1)$$

where r_o and r_i are the fixed inner and outer radii of the device; $\Omega_c = \pi h(r_i^2 + r_c + r_i r_c)/3$ is the volume of fluid in the double-cone chamber; $U_{el} = U_{el}(h, r_c)$ is the elastic energy of the deformed polymer layers; $A_z = \pi(r_o^2 - r_c^2)$ is the area the zipped electrode portion; $t = t(h, r_c)$ is the thickness of a single polymer layer in the zipped region (it varies with h and r_c due to strain); ϵ_p is the dielectric constant of the polymer; and E is the electric field in the zipped region, which depends on the applied voltage V : $E = V/(2t)$. The two terms on the righthand side of both equations in (1) account for the elastic restoring forces due to the dielectric polymer elasticity, and the contribution of Coulomb electrostatic forces. The latter contribution, in particular, holds a term $\epsilon_p E^2$ proportional to the Maxwell stress applied on the polymeric layers over the zipping region.

The response of the EBM in the different working modes can be described in terms of the different applied loads (namely, F or p in Eq. (1)). When the EBM works as a contractile actuator, the fluid dwells at a pressure close to atmospheric value ($p \simeq 0$), and the force F , which makes work in the direction of h , represents the useful output load. When the EBM works as a pump, p represents the useful available pressure, and F is either zero or equal to a constant pre-load. As a generator, the EBM can take advantage of either the input mechanical work provided by the external force (reverse muscle) or by the fluid pressure (reverse pump) to produce an electrical energy output.

Manufacturing

The pouch consists of two layers of dielectric polymer, attached together with a thin annular layer of double-sided adhesive film. The top film has a circular opening serving as the fluid outlet. Stiff reinforcing annular frames are bonded on the outer faces of the dielectric films with the aim of preventing deformations at the outer diameter of the films. A reservoir is connected to the top opening and an end-effector on the bottom side (Fig. 2A). The reservoir includes a manifold which generates a radial fluid flow from the axial movement of the EBM, and a slack membrane that implements a (nearly) constant-pressure variable-volume sealed reservoir. Conductive paint is applied on each side of the pouch (Fig. 2B) to form the electrodes. The device is pre-loaded and filled up with dielectric oil through the central opening (Fig. 2C).

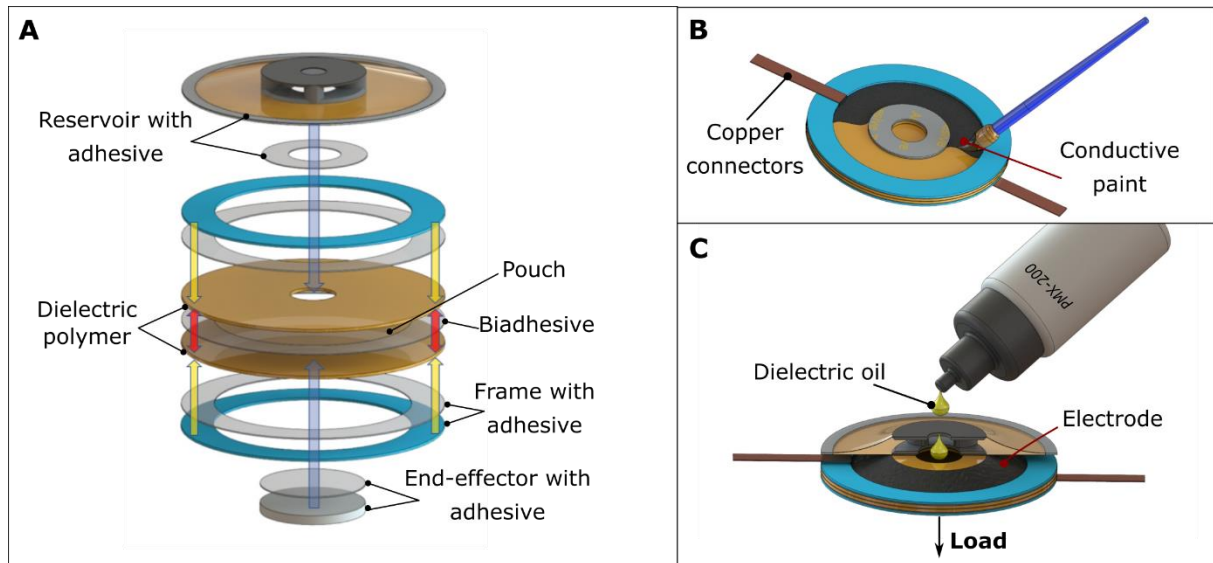


Fig. 2. Layout and fabrication procedure of an EBM unit. (A) Exploded view of the EBM. Red arrows show the bonding of two dielectric layers with a thin adhesive on a circular edge. Blue arrows show the bonding of structural annular frames on the two sides of the pouch. Yellow arrows represent the application of the end-effector and the fluid reservoir. (B) Electrodes are painted on the external surfaces of the dielectric layers with conductive paint. Flexible copper-tape connectors are used to create the connection with the circuit wires. (C) The EBM is filled with dielectric oil through an opening in the reservoir. The opening is sealed after filling.

EBM - contractile actuator

An EBM unit can work as a contractile actuator achieving contractions of up to 20% its initial length (including the length of the fluid reservoir) against external loads in the range 1-6 N (up to roughly 70 times the whole muscle and fluid storage weight) with input voltages in the range 6-8 kV.

At 1 Hz, the EBM unit produced strokes of 1.3 mm (Fig. 3A), corresponding to a contraction of 18.0% the length before actuation, with a maximum applied voltage of 6 kV. Neglecting the contribution of the reservoir height, the stroke corresponds to a contraction of 44.7 %. Fig. 3A reveals that, at 1 Hz, the device sweeps the whole available stroke and remains latched at upper and lower positions for a finite amount of time. The EBM moves synchronously with the applied voltage: when voltage is low, concurrent polymeric layers are kept apart (unzipped) under the action of the mechanical load; as the voltage increases, electrostatic forces progressively overcome the external load and cause the electrodes to zip; at large voltage (roughly above 5 kV), the electrodes stay fully zipped against each other and almost all of the liquid is moved into the reservoir. The EBM experiments a very limited reduction in stroke due to increased frequency (Fig. 3A), similarly to hydraulically-amplified muscles [24] and in

contrast with frequency-sensitive fluid-based electrostatic ribbons [25]. The EBM stroke at 4 Hz and 8 Hz is respectively 97.0% and 92.8% the stroke at 1 Hz, with a load of 1.5 N. Increasing the peak voltage to 8 kV, the EBM reaches a total contraction over 20% with forces of up to 2 N. An overview of the EBM unit response in quasi-static conditions under different applied forces is shown in Fig. 3B. In the low-force range (1-2 N), increasing the applied load leads to an increase in the stroke, as a result of the increased initial length of the muscle. A further increase in the load, which acts against the electrostatic forces, causes a decrease in the actual stroke. Remarkably, the EBM still generates contractions over 17% with loads of up to 3 N, i.e. an equivalent lifting capability of 35 times its total weight (including the weight of the fluid reservoir) and 50 times the EBM core weight. Increasing the peak voltage generates a modest increase in stroke at small loads, whereas it has a significant impact at high loads. In effect, at reduced loads, a relatively low voltage is sufficient to cause a contraction close or equal to the whole geometrically available stroke. Larger voltage still produces larger strokes even at 1-1.5 N, as it enables better fluid evacuation and the permanence of a thinner trapped fluid film in the fully-zipped configuration.

In the range 1-10 Hz, the EBM contraction weakly depends on the frequency even at relatively large applied loads (Fig. 3C). At 2.5 N, the contraction is nearly constant (close to 19%) at 8 kV in the range 1-10 Hz, whereas it slightly decreases with frequency at 6 kV (with the stroke at 10 Hz still being 74.6% that at 1 Hz), since the electrostatic forces do not completely compensate viscous and hydraulic losses. The EBM provides actuation up to frequencies of at least 80 Hz (Fig. 3C) with 2 N of applied load, experimenting a reduction in stroke of 50% at a frequency of 37 Hz and of 80% at 80 Hz. The reduction in stroke at larger frequencies is ascribable to viscous losses due to the dielectric fluid motion.

The blocking force of the EBM (at which the stroke falls to zero) at 8 kV is close to 6 N (i.e., 98 times the weight of the EBM core), rather independently of the actuation frequency (Fig. 3D). The stroke at 5 N is roughly 65% the maximum stroke, and it drastically falls in the interval 5-6 N.

The EBM unit achieves contraction rates of up to 1140 %s⁻¹ at 10 Hz (Fig. S8), with power densities over 21 W/kg per EBM core mass (which still corresponds to 14.8 W/kg if the weight of the fluid reservoir is also considered).

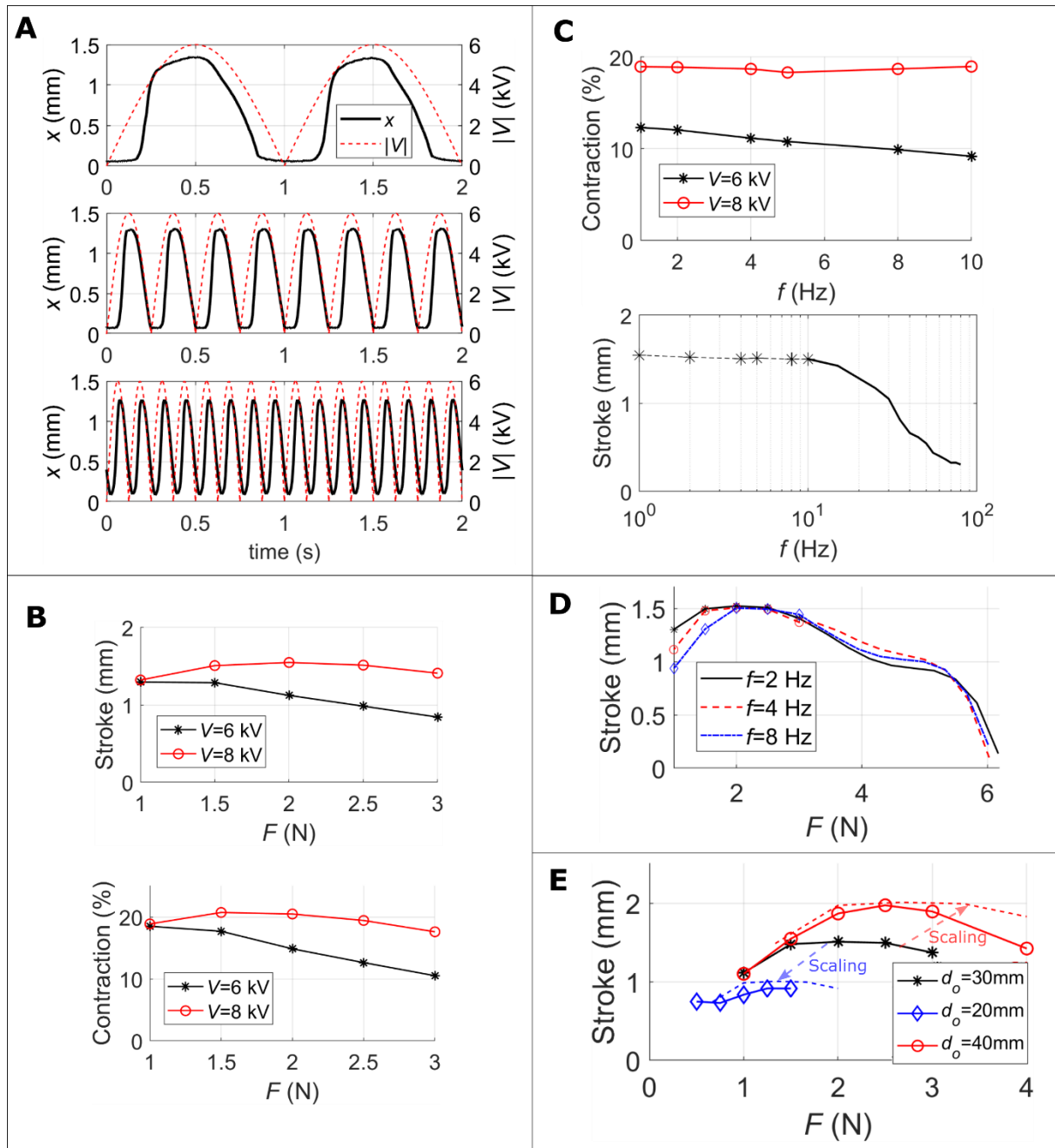


Fig. 3. Force-contraction response of an EBM unit. (A) Time-series of the voltage V and actuator displacement x at 1 Hz (top), 4 Hz (middle), and 8 Hz (bottom) with a load of 1.5 N. The voltage module is shown for clarity, though in practice its polarity is reverted at each cycle. (B) Force-stroke (top) and force-contraction responses at 1 Hz (bottom), with different applied forces F and voltages. (C) Frequency response at different voltages, with 2.5 N load (top). Markers refer to tests with different frequencies. Results of maximum-frequency test with 2 N load and 8 kV voltage (bottom). The dashed line with markers refers to constant-frequency tests, whereas the continuous line refers to a test with chirp voltage input. (D) Blocking force test at different frequencies with a voltage of 8 kV. Markers refer to constant-force, whereas line extensions refer to tests with progressively increasing force. (E) Force-stroke response of three different EBMs: the base-case device (outer diameter $d_o=30$ mm), and two versions with radial dimensions scaled by $s_f=2/3$ ($d_o=20$ mm) and $s_f=4/3$ ($d_o=40$ mm) respectively. Markers represent constant-force constant-frequency measurements (4 Hz, 8 kV), whereas dashed lines represent theoretically scaled responses with respect to the base-case scenario.

The EBM has also achieved interesting performance in terms of long-term cyclic operation. An EBM prototype subject to electric fields in the order of 120 kV/mm on the dielectric polymer layers was able to accomplish more than 0.1 Mcycles at 5 Hz operating frequency, with no visible reduction in stroke over time (Fig. S11).

Scaling the EBM response

We investigated the effect of scaling the radial dimensions of an EBM contractile actuator while keeping the thickness of the dielectric polymer layers constant. According to the proposed mathematical model, a geometrically scaled device (with geometric scaling factor of s_f) loaded with a force scaled by s_f provides a stroke that is s_f times larger, at a same voltage (derivation provided in SM section A.3).

The experimental data on three different devices (i.e., the base-case device to which previous results refer, a smaller scale device with $s_f = 2/3$, and a larger device with $s_f = 4/3$) confirm the trends predicted by the model (Fig. 3E and Fig. S9). Observed disagreements are possibly due to the lack of stiffness for the upscaled device and a higher sensitivity to imperfect manual manufacturing of downscaled devices.

Reducing the scale of an actuator leads to an increase in the achievable energy density (see Sect. A.3 in the SM), due to the significant reduction in the dielectric fluid mass (which scales with s_f^3 , as opposed to the actuation work, which scales with s_f^2). A downscaled EBM prototype with lightweight polymeric structural components was able to achieve an energy density (i.e., cyclic actuation work per unit EBM core mass) of 1.16 J/kg subject to a load of 1.6 N, i.e., 127 times its own weight (Fig. S10).

EBM arrays from foldable strips for scaling up force and stroke

EBM units can be combined in-series and in-parallel to magnify the stroke and the force respectively, while preserving the ease of manufacturing and assembly of single unit AMs. In-series connection is achieved by stacking up multiple devices [32], [33] and putting their fluid chambers in communication. In-parallel connection is achieved by providing a set of EBM units/stacks with a common end-effector. By connecting in parallel m stacks of n in-series EBMs, we obtain $m \times n$ arrays that provide a stroke which is n times magnified, in the presence of an applied force which is m times as much.

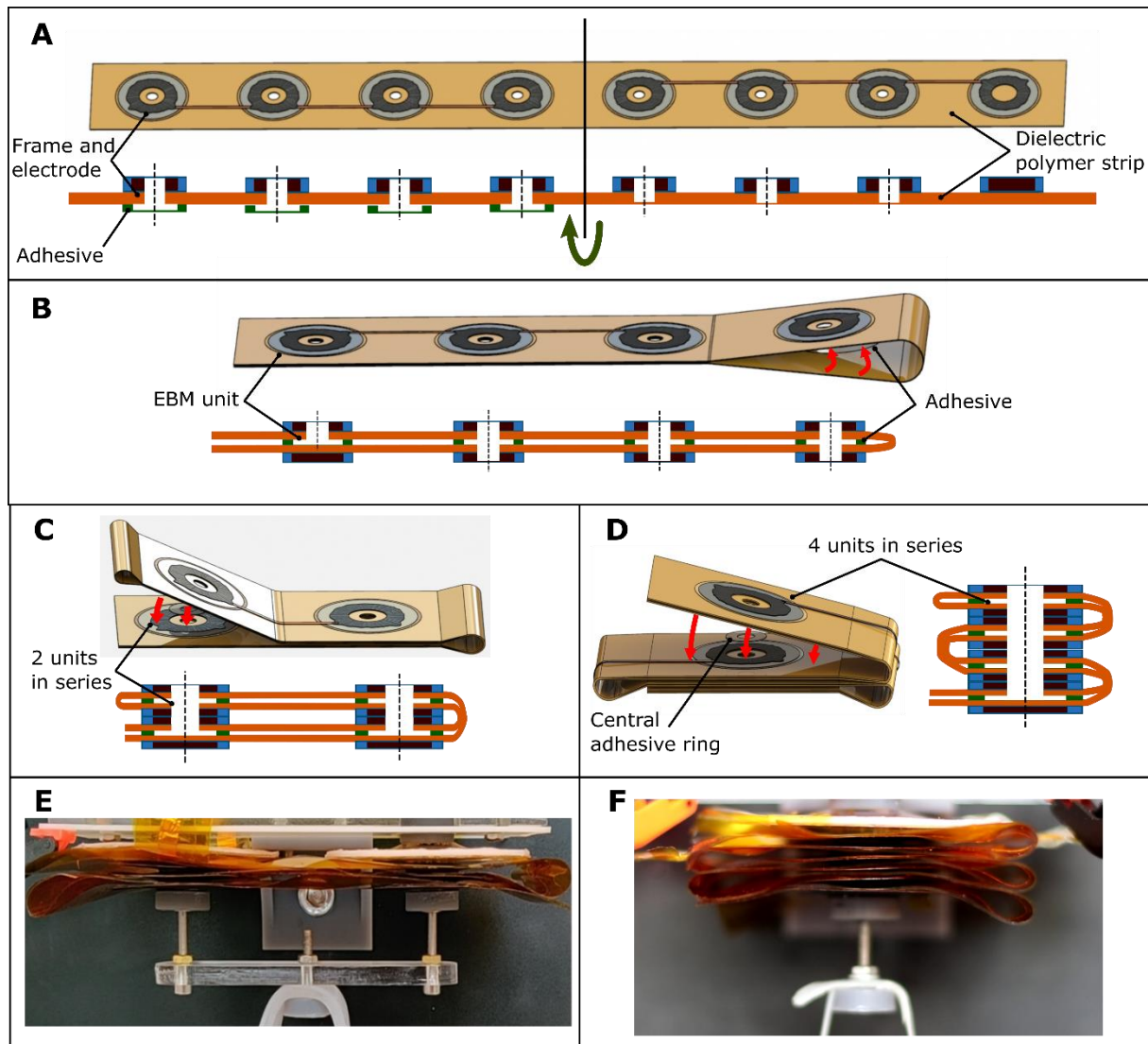


Fig. 4. Folding procedure of dielectric polymer strips to fabricate EBM arrays. (A) Dielectric polymer strip with frames, electrodes and adhesive tape attached, represented both in 3D and cross-section view. The black vertical line represents the first fold. (B) 3D and cross-section views of the four parallel units created by the folding procedure. (C) 3D and cross-section views of a 2×2 EBM array obtained by folding. In-series units are attached together around the perimeter of their adjacent openings with annular-shaped double-sided adhesive film. (D) 3D and cross-section views of a 1×4 EBM array obtained by folding. (E) Photograph of a 2×3 EBM array. (F) Photograph of a 1×6 EBM array.

Different array layouts can be built through a same manufacturing procedure (Fig. 4 A-D and Fig. S4). The constitutive elements of the EBM units (electrodes, central openings and frames with adhesives) are mounted equidistantly and collinearly on a strip of dielectric

material, forming a row of “half-units” on one surface of the strip. Fig. 4A shows such a strip in the case of four-unit array. This strip is then folded in half (with the frames and the electrodes on the external faces) to create EBM units (Fig. 4B). In the same way as single EBM units, each couple of half-units is attached and sealed together with thin layers of double-sided adhesive film. The resulting strip of assembled single units can be folded again to create arrays with arbitrary size. Fig. 4C shows folds required to create a 2×2 EBM array, where a strip with two units is folded onto another one of the same size. In Fig. 4D, the same initial strip is folded in a different way so as to create a 1×4 array.

Starting from a strip of p EBM units, arrays of different sizes can be created, obtaining a maximum blocking force equal to p times that of the single units (in the case of a $p \times 1$ array) or a maximum stroke equal to p times that of a unit (in the case of a $1 \times p$ array). To create an $m \times n$ array from a strip of p units (with $p = m \cdot n$), $n - 1$ folds are required. The simplicity of this assembly procedure makes it scalable and well suitable for mass production.

Performance of contractile EBM arrays

We built EBM arrays with different configurations, comprising up to 6 pouches each (Fig. 5A), with the same reference dimensions for the units as those used in Fig. 3A-D. By connecting in-series six EBM pouches (Fig. 5B) in a 1×6 arrangement, we reached strokes of up to 43% the muscle initial length (including the height of the fluid reservoir). We characterized the response of in-series arrays with loads of up to 3 N (20 to 40 times their weight). We compared the performance of 1×3 and 1×6 arrays against that of a single unit device (Fig. 5C): at 8 kV, the stroke against a force of 1.5 N surpassed 4.1 mm for a 1×3 device (31.0% contraction), and 9.0 mm for a 1×6 array (42.8% contraction), as compared to 1.5 mm for the unit device (20.2% contraction). By increasing the number of in-series units, the height of the fluid reservoir (which is the same for all devices) becomes increasingly small compared to the total stroke, leading to an increase in the percentage contraction. Reducing the fluid reservoir height would generate an increase between 10 and 30 % in the contraction for the different arrays. For instance, at 1N, the estimated contraction for the 1×6 EBM would rise to 55.2% (as compared to 43.3%) if the reservoir height was virtually reduced to zero.

We compared the dynamic responses of different in-series EBM units (Fig. 5D and Movie S1). Similar to the unit actuator, in the range 1-10 Hz the contraction achieved by EBM arrays

is weakly dependent of the frequency. Compared to single and 1×3 devices, 1×6 arrays experiment larger reductions in contraction at large frequencies (above 5 Hz). This is due to dynamical effects (e.g., the increase of inertia with the number of units) and the higher sensitivity to manufacturing/misalignment errors of large EBM arrays.

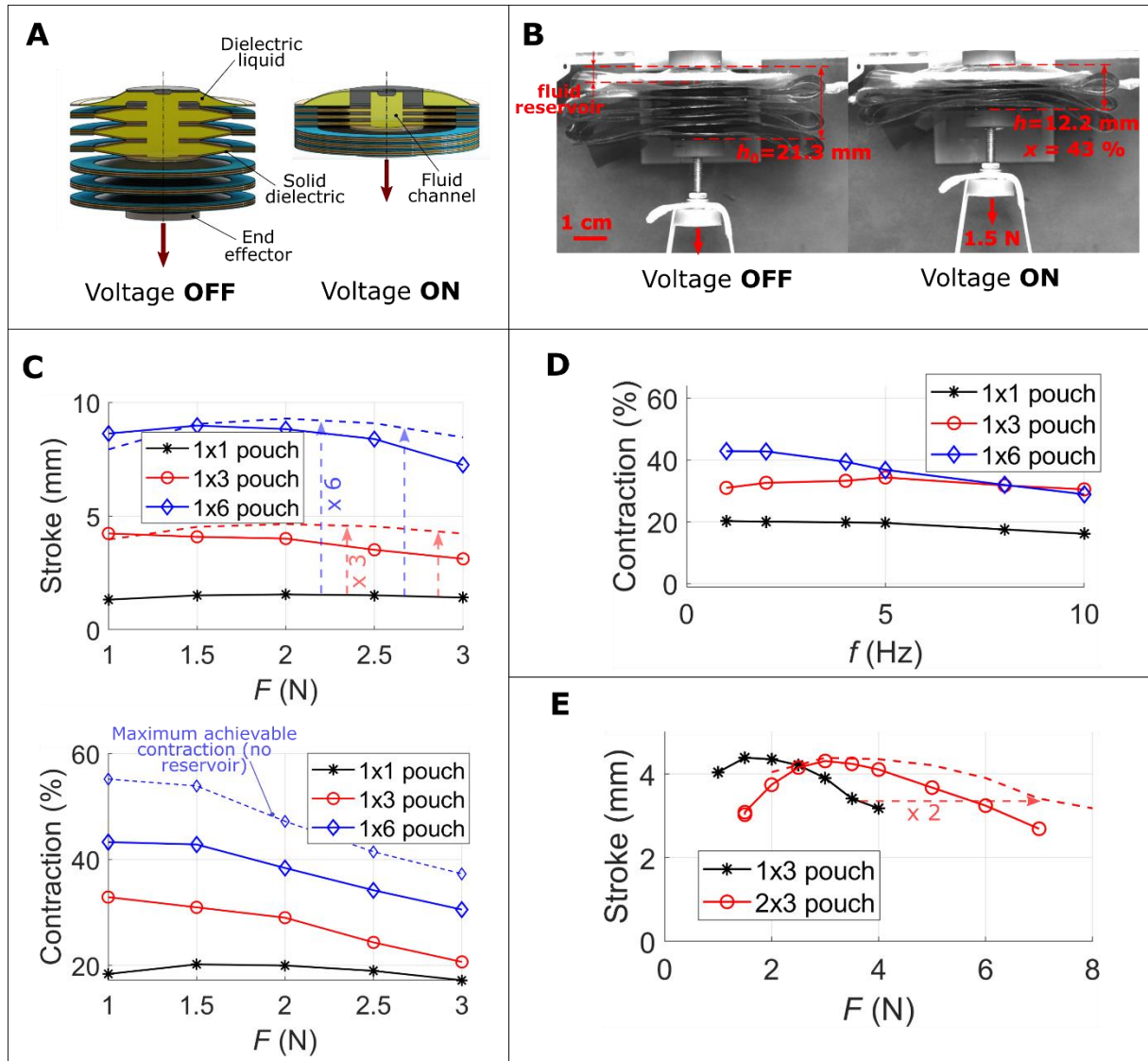


Fig. 5. Performance of arrays of EBM units. (A) Schematic of a 1×6 array. (B) A 1×6 array subject to a voltage of 8 kV produces a stroke of 42% against a 1.5 N load at 1 Hz. (C) Comparison of the quasi-static force-stroke response of 1×1 , 1×3 and 1×6 EBM arrays subject to 8 kV actuation cycles at 1 Hz. The top plot compares the absolute strokes, whereas the bottom one reports the percentage contraction. Dashed lines in the top plot represent the stroke of a unit pouch multiplied by 3 or 6, as compared to the stroke of the arrays. The dashed line in the bottom plot represents the achievable contraction in case the fluid reservoir height was reduced to zero. (D) Frequency response of 1×1 , 1×3 and 1×6 arrays subject to 1.5 N and 8 kV. (E) Comparison of the response of 1×3 and 2×3 EBM arrays subject to 8 kV actuation at 4 Hz. The dashed line represents the response of the 1×3 device scaled by 2 along the force axis, as compared to the response of the 2×3 device.

EBM arrays of in-series units were able to reach peak linear speeds close to 200 mm/s (see Fig. S13) and contraction over 1250 %s⁻¹ (see Fig. S12).

We measured peak specific power of 31 W/kg (see Fig. S12) for the 1 × 3 actuator, in the same order as other fluid-based electrostatic actuators [24] , [25]. Shape-memory alloys and coiled nylon AMs reach higher power densities of tens of W/g [14], [34]. Nonetheless, they suffer from poor controllability at large frequencies [24], due to their slow thermal dynamics.

We developed a 2 × 3 EBM array, made of two in-parallel 1 × 3 arrays (Fig. 4E). As expected, this device produces a stroke close to that of a 1 × 3 array in the presence of a load which is twice as much (Fig. 5E, Fig. S14 and Movie S2). In practice, scaling the force-stroke response of a 1 × 3 EBM array of a factor 2 along the force axis leads to a slight overestimation of the 2 × 3 array stroke. This is due to a higher sensitivity of the 2 × 3 device to bending moments generated by the total load, applied between (and roughly parallel to) the axes of the in-parallel stacks. We used the 2 × 3 EBM array to lift large weights (see Movie S2). The device can lift hanged masses as high as 1000 g, i.e. over 40 times its weight. This is consistent with the results on single unit devices (Fig. 3D), which showed blocking forces for single units (and, virtually, in-series arrays) close to 6 N.

EBM - volumetric pump

EBM arrays can be employed as volumetric pumps capable to operate FASs. We employed a 1 × 6 EBM with reference dimensions to inflate a thin elastomeric membrane with 45 mm base diameter, and we reached operating pressures of up to 6 kPa and flow rates over 0.5 SLPM.

The EBM-based volumetric pump was obtained putting the dielectric liquid chamber in communication with the air chamber that holds the membrane, and using two check valves to produce unidirectional flow (Fig. 6A), similarly to [28]. With this layout, each stroke of the dielectric liquid free surface generates a useful unidirectional air flow from the atmosphere towards the bubble. By performing tests at 7 Hz with a voltage of 8 kV and a pre-load of 1.5 N, we were able to inflate the balloon up to 56 cm³ in 28 s (Fig. 6B and Movie S3). By performing tests at different frequencies, we observed that the pumped volume increases nearly proportionally with the frequency. As a result, the trend of the pumped volume and pressure vs the number of actuation cycles is approximately the same at operating frequencies of 5 and 7 Hz (Fig. 6B). At low frequency (i.e., 3 Hz), the pressure and air volume delivered after a same

number of cycles are lower. This is possibly due to air leaks in the pump body, which are independent of the pumping frequency and are not counterbalanced by a sufficient delivered flow at low operating frequencies. The pump flow rate (averaged on the single actuation cycles) as a function of the hydrostatic pressure has the trend shown in Fig. 6C. In the low-pressure range (below 2 kPa), the EBM pump delivers a flow in the range 0.40-0.63 SLPM, which increases with the frequency. In these conditions, the EBM sweeps a stroke close to 5 mm (corresponding to a volumetric displacement of 2.1 cm^3) and delivers a virtually constant volumetric displacement. By measuring the bubble volume variation within a cycle (see lens in Fig. 6B), we estimated a maximum instantaneous flow rate of 1.95 SLPM (Fig. S17B). As the hydrostatic pressure increases, the EBM stroke and, hence, the flow rate decrease. As expected, at given pressure, the delivered flow rate increases with the actuation frequency following a nearly linear trend (Fig. 6D). The blocking pressure, above which the pump is no longer able to supply a flow, is slightly larger than 6 kPa, both at 5 and 7 Hz. At lower actuation frequency, the blocking pressure decreases, due to air leaks (Fig. 6C). At the blocking pressure, the EBM still exhibits a stroke of 2 mm, even though the supplied flow is equal to zero. This is due to air leaks and the effect of air compressibility in the pump body.

The EBM was able to develop a pumping power (cycle average) of up to 18 mW (corresponding to a power density over 0.9 W/kg per unit EBM core mass), and instantaneous peak values of 68 mW (Fig. S17).

The performance achieved by the EBM pump is comparable to that of best performing smart-material based FASs with similar dimensions. Similar figures of merit have only been achieved by state-of-the-art DE driven pumps, exploiting snap-through instability of inflatable membranes [35] or a dynamic resonant design [28], [36]. Compared to the first solution [35], the EBM pump allows a significant reduction in encumbrance and offers the potential for compact designs. Compared to resonant DE FAS [28], the EBM pump achieves large flow rates with lower actuation frequency (below 10 Hz) and with no stringent need to tune the operating frequency. At a reference pressure of 2 kPa, our prototype generated a flow rate in the same order as that of a DE pump with similar radial dimensions [36], with frequencies one order of magnitude lower.

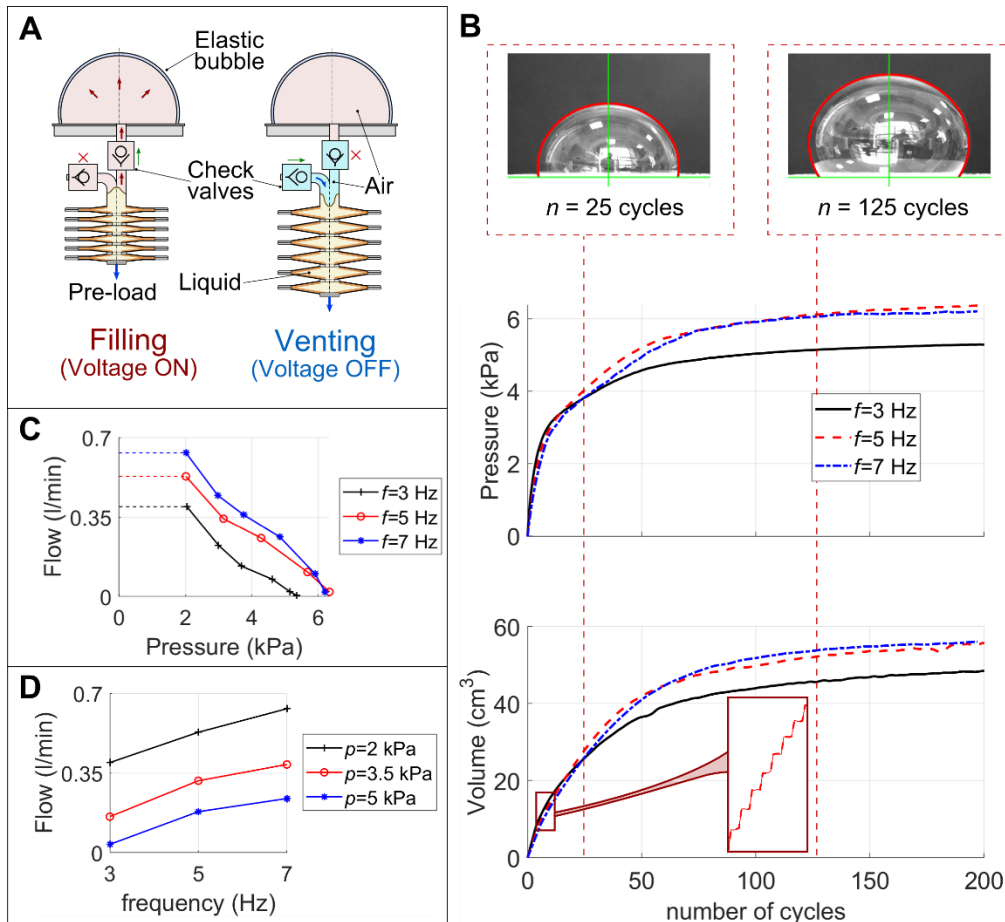


Fig. 6. Characterization of EBM-driven FAS. (A) Operating principle of the EBM-driven pump. The EBM output is connected to an air chamber holding two check valves. The dielectric liquid free surface acts as the pumping element. Working cycles consist of a venting phase and a filling phase through which air is unidirectionally transferred from the atmosphere to the air chamber. (B) Trends of the pressure and the pumped volume as a function of the number of pumping cycles, at different frequencies (3, 5 and 7 Hz). The top boxes show the progression of the balloon expansion, whereas the lens shows the step-wise volume increase for the test at 5 Hz (each step corresponds to a single stroke of the EBM). (C) Pressure-flow diagram of the EBM pump at different frequencies. The pressure increase from 0 to 2 kPa only spans a few actuation cycles, therefore the trend of the flow over the interval 0-2 kPa is replaced with its average value (dashed lines). (D) Flow rate as a function of the frequency, at three different pressures.

EBM - energy harvester

EBM arrays can function as generators with no need for architectural modifications compared to the actuator embodiment. To demonstrate generator operation, we supplied mechanical energy to a lightweight 1×3 EBM with polymeric structural rings by connecting the end-effector to the slider of a linear motor via a pulling nylon cable, and we controlled the applied voltage synchronously with the motor displacement (Fig. 7A). We applied a constant voltage while the motor was pulling the EBM's end-effector downwards (pushing the

electrodes apart) and kept the device uncharged while the motor was moving upwards (allowing the electrodes to move closer) (Fig. 7A). By doing so, an amount of electrical energy is initially spent when voltage is applied; then, as the external mechanical load forces opposite electrodes to be pushed apart, the EBM positively delivers current, leading to a net gain of electrical energy. The electrical energy balance is depicted in Fig. 7B, which relates the EBM voltage to the total charge delivered by the power supply [30]. Fig. 7C shows the EBM force against the end-effector position: due to voltage application, the force required to pull the electrodes apart (forward stroke) is larger than that required to allow the fluid discharge (return stroke), i.e., a net amount of mechanical work is supplied to the EBM at each cycle. Using different values for the priming voltage, we obtained a cyclic generated energy density of up to 0.41 J/kg (per EBM core mass), average power of up to 1.5 mW, and maximum efficiency close to 20 % (Fig. 7D). The peak power output (for the test in Fig. 7A) is 80 mW, corresponding to a peak power density of 11.2 mW/g. These figures of merit increase monotonically with the priming voltage. In contrast with theoretical trends for electrostatic generators [30], the power and energy density have a sub-quadratic trend, possibly due to an increase in the leakage currents with the voltage.

As compared to DE generators, the stored elastic energy (due to mechanical loading) for EBMs is on the same order as the maximum stored electrostatic energy, while in DE generators it is 3 to 50 times the stored electrical energy [37]. This is a distinguishing feature that makes it easier coupling EBM harvesters with mechanical energy sources without the need of stiffness cancellation mechanisms (e.g., negative springs) [38], [39].

The values of energy and power densities could be largely improved to get closer to the values achieved by DE generators (i.e. energy densities on the order of a hundreds of J/kg and power densities in the order of hundreds of W/kg [40], [41]). Due to the low elastic energy stored in EBMs, mechanical losses are lower than in DE generators, nonetheless, charge leakages are estimated to be significant. Fig. 7B shows that the charge absorbed by the EBM (which differs from the real charge on the electrodes due to leaks) is significantly larger than zero by the end of the cycle (state P_4), meaning that some current has been drained. These losses appear attributable to leaks in air (since the outer surfaces of the electrodes are not shielded) and through the dielectric layers, with the former probably being more relevant. Introducing shielding of electrodes and using different dielectric liquids might be the key to enhance EBM generators performance. In addition to that, the optimization of applied voltage waveform [40],

duration of the charging/discharging phases and resorting to close-loop logics is expected to lead to further improvement.

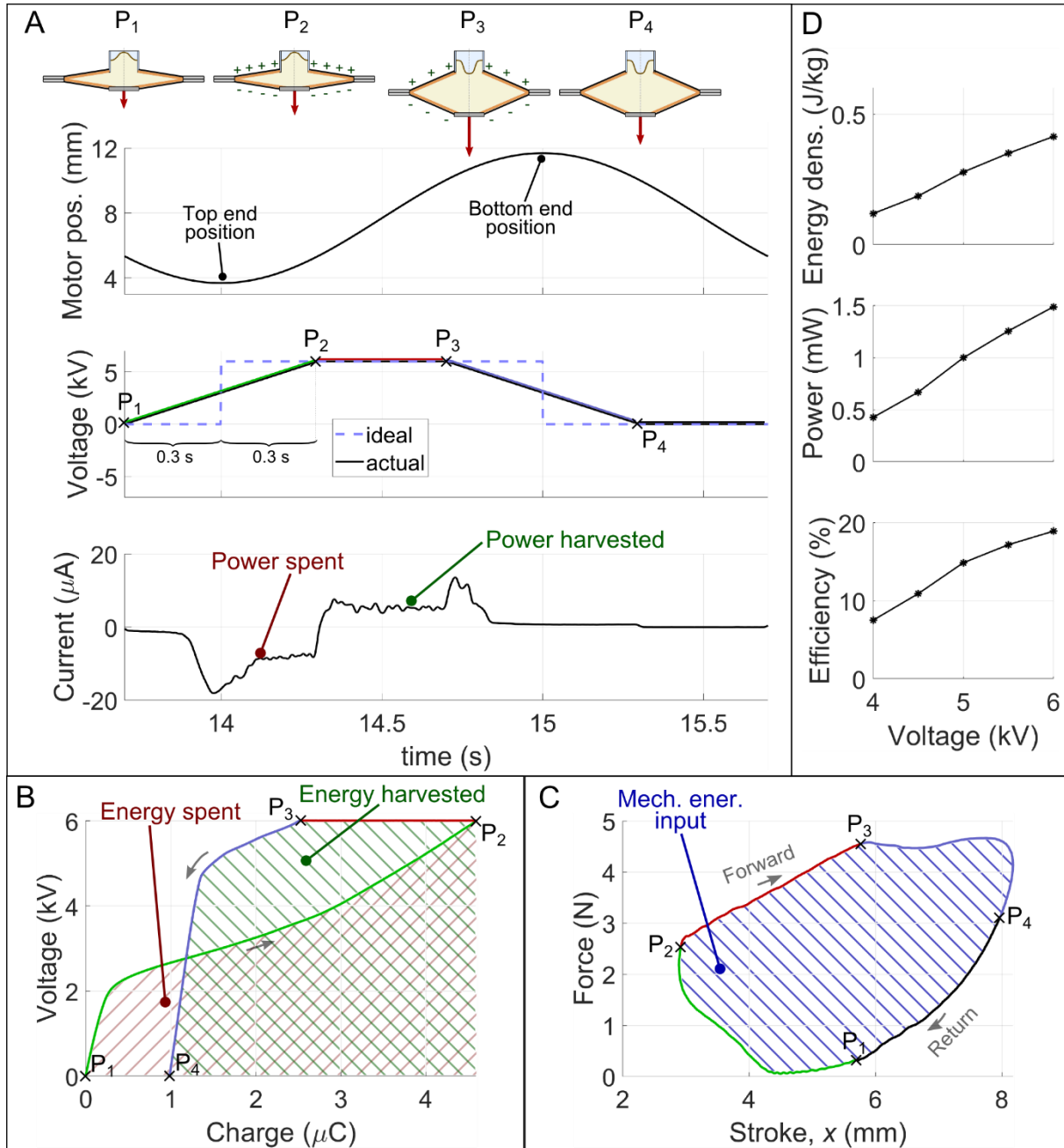


Fig. 7. EBM energy harvester characterization. (A) Time-series illustrating the working cycle of the EBM generator. Top: input motor position (positive values represent downward displacement with respect to the EBM fully-zipped configuration). Middle: EBM voltage (here, maximum voltage is 6 kV). Bottom: Output current of the EBM. Ideally, the generator is held at constant voltage while the stroke increases, and uncharged while the stroke decreases (dashed curve). In practice, charging (discharging) is started 0.3 s before and concluded 0.3 s after the motor has reached the top (bottom) end of its stroke, in order to limit currents and losses. (B) Applied voltage vs. supplied charge trajectory and (C) force vs. end-effector stroke trajectory for the test of the previous sub-figure (the stroke x is equal to zero when the EBM units are flat). The hatched areas in the voltage-charge plane represent the electrical energy spent for priming and that positively supplied by the generator. The hatched

area on the force-stroke plane equals the input mechanical energy. **(D)** Generated energy density, mean power, and mechanical-to-electrical efficiency as a function of the applied voltage.

Discussion

In this paper we introduce a class of cheap easy-to-manufacture actuators, called electrostatic bellow muscles (EBM) which hold the ability of implementing different operating modes through a single device layout. Besides working as linear actuators that contract on activation, EBMs can also operate as hydrostatic pump for fluidic actuators, or as energy harvesters. For each of these modes they show performance that is comparable to that of other promising single-purpose electrostatic actuators [21], [24], [25]. Straightforward magnification of EBMs displacements and forces can be obtained through stacking and parallelization of units with a simple folding process, which is extremely promising for mass-production.

As contractile actuators, EBM arrays reach contractions of up to 43%, i.e., more than two times the typical contractions of skeletal muscles [8]. EBM AMs reach performance on the same order as other recently developed fluid-based electrostatic actuators. Compared to Peano-HASEL actuators, the EBM reaches larger contractions, with a moderately lower power density [24], [27]. Compared to electro-ribbons, EBMs reach smaller contractions, but a significantly larger strain rate, while guaranteeing a tighter encapsulation of the dielectric fluid [25]. As hydrostatic drivers for FASs, cm-scale EBMs can generate pressures of a few kilopascals (comparable or better than other smart material-driven pump prototypes [5], [36]), and flow rates on the order of ~ 1 SLPM (larger than those of existing smart material based pump prototypes, and on the same order as commercial micro-compressors [4]). As energy harvesters, thanks to their high generated-to-elastic energy ratio, they can generate electrical power with efficiencies of up to 20%.

Being lightweight, scalable and adaptable to different tasks, the EBM appears as a promising option for robotic systems (e.g., soft robots, lightweight autonomous robots), which are required not only to perform a diversity of tasks but also, in perspective, to be able to harvest their energy [43].

In the future, the use high-permittivity high-strength dielectric polymers and the miniaturization of EBM units might improve power and energy densities and reduce required operating voltages. In generation mode, the use of fluids with better dielectric properties and an improved encapsulation of the electrodes seems a key factor to further improve efficiency.

Materials and Methods

EBM materials and components

The dielectric layers in EBM are made of 25 μm -thick LINQTAPE PIT1N polyimide (PI) film by Capling, with nominal relative permittivity of 3.9 and dielectric strength of 236 kV/mm [44]. The dielectric fluid is Xiameter PMX 200 50 cSt, a silicone oil with relative permittivity of 2.7 [23].

Bonding of the dielectric layers on the pouch inner surfaces is made with 5 μm biadhesive tape by Nitto. Rigid rings are attached to the dielectric via 100 μm thick biadhesive tape by Tesa. We produced two different versions of the EBM based on different materials for the structural rings: a version with thin rigid metallic frames (to grant the device with a reduced initial length, and hence achieve larger contractions), and one with lightweight PMMA rings (to reduce the weight and increase the energy density). Metallic annular frames made of steel, with a thickness of 0.1 mm were used for the actuator and pump tests. PMMA polymeric frames with 0.5 mm thickness were used for generator tests, although some actuation tests were also performed on PMMA-based EBMs (Fig. S10).

Electrodes (16 x 30 mm in the base-case device) are made of DAG-T-502 carbon paint by Ted Pella. This paint shows appropriate compliance and good adhesion to the substrate. To attach pouches in-series in arrays, and to attach the reservoir or the end effector we used 0.5 mm thick 3M™ VHB™ 4905 double-sided adhesive tape.

The fluid reservoir contains three main components: a rigid layer, a manifold (which controls the direction of the flow within the reservoir) and a flexible membrane. The rigid layer consists of a PI film, reinforced by a 0.5 mm thick PMMA annular frame. The manifold is 3D printed with a Formlabs printer with Grey Pro photopolymer resin. The flexible membrane is a loosely stretched layer of 3M™ VHB™ 4905, which grants the reservoir with variable volume, while producing negligible pressurization in the fluid. The total height of the reservoir is 4.3 mm, whereas its mass is 2.6 g.

EBM prototypes with different radial sizes were built. A base-case reference set of dimensions were used for most of the reported tests (Fig. 3A-D, Fig. 5, Fig. 6, Fig. 7). The reference EBM has outer electrode diameter $d_o = 30$ mm, and was either realized with 30 x 42 mm steel rings (Fig. 3A-D, Fig. 5, Fig. 6) or 30 x 40 mm PMMA rings (Fig. 7). A complete set of reference dimensions for EBM unit prototypes are summarized in Fig. 2A, Fig. S5 and Tab.

S3. Smaller or larger scale prototypes with thin steel rings (20 x 32 mm and 40 x 52 mm) or PMMA rings (20x27 mm) were also realized to study the scaled response of AM EBMs (Fig. 3E). The dimensions and weights of the different prototypes employed for the tests are summarized in Tab. S3.

Testing methods

A test-bench was developed and adapted to characterize EBMs (unit and stacks) in different working modes (see Fig. S6). High-voltage was supplied by means of a four-quadrant amplifier (TREK 10/10B-HS). We used voltages of up to 8 kV, corresponding to a maximum electric field of 160 kV/mm over the total polymer layers thickness, i.e., well below the nominal breakdown field of the reference PI material (236 kV/mm).

Driving, control and data acquisition were performed with a real-time target machine by Speedgoat, running Matlab Realtime environment. The EBM end-effector stroke was measured via a laser sensor (LK-G152 by Keyence). A protruding structure was attached to the device end-effector in order to facilitate load application and stroke measurement.

Linear muscles testing

Each linear actuation test was performed applying a constant force on the end effector, supplying a time-varying voltage profile on the electrodes, and measuring the resulting displacement. Constant forces at the EBM end-effector were generated through a pre-loaded soft elastic band with initial length much larger than the actuator stroke. We used sinusoidal voltage profiles, thus reverting the electrodes polarity at each cycle to compensate for polarization of the interfaces between different dielectric layers, as suggested in literature [24], [45]. The driving voltage frequency is one half the resulting actuation frequency. In our results, the percentage contraction (see Eq. (S15)) is defined as the ratio of the stroke and the EBM length before voltage application at the tested load case (average over the performed actuation cycles). The latter is measured from the top surface of the reservoir to the bottom layer surface (where the end effector is attached). The thickness of the end-effector is not taken into account, since its layout is only dictated by experimental convenience, and it can be easily redesigned and optimized.

EBM pump testing

In hydrostatic actuation tests, the EBM fluid outlet was connected to a 3D printed assembly holding custom-made check valves and the VHB membrane bubble. The inflatable membrane

was realized using 3 layers of 500 μm thick commercial 3M™ VHB™ 4905, with a pre-stretch of approximately 2.5. In the tests, we preloaded the EBM with a constant force of 1.5 N, and we used sinusoidal voltage inputs with frequency in the range 1.5–3.5 Hz (corresponding to 1/2 of the resulting actuation frequency). The pressure in the air chamber holding the elastic bubble was measured with pressure sensor MPX12 by Freescale Semiconductor. The bubble volume was estimated via the frames of a high-speed camera (Point Grey GS3-U3-23S6M-C with lens 250F6C). Standard edge detection algorithms were used to resolve the bubble profile, similar to [41], and the subtended volume was calculated by numerical integration.

Energy harvester testing

To test the EBM generator, we drove the EBM displacement with linear motor L16-50-35-12-P by Actuonix. The connection between the slider and the EBM end-effector was implemented via a fluorocarbon fishing wire (300 μm diameter). Compared to a rigid connection, this ensures a smoother deformation of the EBM and rejects undesired transversal loads, although it does not grant direct drivability of the end-effector position. Similar to actuation tests, the voltage polarity was reverted at any cycle. In the generation tests, we used motor displacements with amplitude of 4 mm, bias of 7.7 mm, frequency of 0.5 Hz. The input mechanical energy was calculated from the end-effector displacement and the force (measured via a load cell Rb-Phi-118 with RB-Onl-38 amplifier by RobotShop) (see Eq. (S16)). The net generated energy was calculated from the voltage (measured with the amplifier's integrated sensor) and the current (measured with isolation amplifier HCPL-7800A/HCPL-7800 in series to the EBM) (see Eq. (S16)).

References and Notes

- [1] D. Rus and M. T. Tolley, “Design, fabrication and control of soft robots,” *Nature*, vol. 521, no. 7553, pp. 467–475, May 2015, doi: 10.1038/nature14543.
- [2] E. Brown *et al.*, “Universal robotic gripper based on the jamming of granular material,” *Proc. Natl. Acad. Sci.*, vol. 107, no. 44, pp. 18809–18814, Nov. 2010, doi: 10.1073/pnas.1003250107.
- [3] B. Mosadegh *et al.*, “Pneumatic Networks for Soft Robotics that Actuate Rapidly,” *Adv. Funct. Mater.*, vol. 24, no. 15, pp. 2163–2170, Apr. 2014, doi: 10.1002/adfm.201303288.
- [4] M. Wehner *et al.*, “Pneumatic Energy Sources for Autonomous and Wearable Soft Robotics,” *Soft Robot.*, vol. 1, no. 4, pp. 263–274, Dec. 2014, doi: 10.1089/soro.2014.0018.
- [5] V. Cacucciolo, J. Shintake, Y. Kuwajima, S. Maeda, D. Floreano, and H. Shea,

- “Stretchable pumps for soft machines,” *Nature*, vol. 572, no. 7770, pp. 516–519, Aug. 2019, doi: 10.1038/s41586-019-1479-6.
- [6] G. K. Klute, J. M. Czerniecki, and B. Hannaford, “Artificial Muscles: Actuators for Biorobotic Systems,” *Int. J. Rob. Res.*, vol. 21, no. 4, pp. 295–309, Apr. 2002, doi: 10.1177/027836402320556331.
- [7] J. D. W. Madden *et al.*, “Artificial Muscle Technology: Physical Principles and Naval Prospects,” *IEEE J. Ocean. Eng.*, vol. 29, no. 3, pp. 706–728, Jul. 2004, doi: 10.1109/JOE.2004.833135.
- [8] S. M. Mirvakili and I. W. Hunter, “Artificial Muscles: Mechanisms, Applications, and Challenges,” *Adv. Mater.*, vol. 30, no. 6, pp. 1–28, 2018, doi: 10.1002/adma.201704407.
- [9] S. Solano-Arana, F. Klug, H. Mößinger, F. Förster-Zügel, and H. F. Schlaak, “A novel application of dielectric stack actuators: a pumping micromixer,” *Smart Mater. Struct.*, vol. 27, no. 7, p. 074008, Jul. 2018, doi: 10.1088/1361-665X/aac302.
- [10] A. Jahanshahi, F. Axisa, and J. Vanfleteren, “Fabrication of an implantable stretchable electro-osmosis pump,” Feb. 2011, vol. 7929, p. 79290R, doi: 10.1117/12.873812.
- [11] O. C. Jeong, S. W. Park, S. S. Yang, and J. J. Pak, “Fabrication of a peristaltic PDMS micropump,” *Sensors Actuators A Phys.*, vol. 123–124, pp. 453–458, Sep. 2005, doi: 10.1016/J.SNA.2005.01.035.
- [12] E. Thielicke and E. Obermeier, “Microactuators and their technologies,” *Mechatronics*, vol. 10, no. 4–5, pp. 431–455, Jun. 2000, doi: 10.1016/S0957-4158(99)00063-X.
- [13] P. Miaudet *et al.*, “Shape and temperature memory of nanocomposites with broadened glass transition,” *Science (80-.)*, vol. 318, no. 5854, pp. 1294–1296, 2007, doi: 10.1126/science.1145593.
- [14] C. S. Haines *et al.*, “Artificial muscles from fishing line and sewing thread,” *Science (80-.)*, vol. 343, no. 6173, pp. 868–872, 2014, doi: 10.1126/science.1246906.
- [15] Y. Bar-Cohen and Q. Zhang, “Electroactive Polymer Actuators and Sensors,” *MRS Bull.*, vol. 33, no. 3, pp. 173–181, Mar. 2008, doi: 10.1557/mrs2008.42.
- [16] Q. Pei, J. Joseph, R. Kornbluh, and R. Pelrine, “High-Speed Electrically Actuated Elastomers with Strain Greater Than 100%,” *Science (80-.)*, vol. 287, no. 5454, pp. 836–839, 2000, doi: 10.1126/science.287.5454.836.
- [17] M. Duduta, E. Hajiesmaili, H. Zhao, R. J. Wood, and D. R. Clarke, “Realizing the potential of dielectric elastomer artificial muscles,” *Proc. Natl. Acad. Sci.*, vol. 116, no. 7, pp. 2476–2481, Feb. 2019, doi: 10.1073/PNAS.1815053116.
- [18] G. Kofod, “The static actuation of dielectric elastomer actuators: how does pre-stretch improve actuation?,” *J. Phys. D. Appl. Phys.*, vol. 41, no. 21, p. 215405, Nov. 2008, doi: 10.1088/0022-3727/41/21/215405.
- [19] M. Sadeghi, H. Kim, and K. Najafi, “Electrostatically driven micro-hydraulic actuator arrays,” 2010.
- [20] H. Kim and K. Najafi, “Electrostatic hydraulic three-way gas microvalve for high-pressure applications.,” in *12th International Conference on Miniaturized Systems for Chemistry and Life Sciences (MicroTAS 2008)*, 2008, pp. 369–371.
- [21] E. Acome *et al.*, “Hydraulically amplified self-healing electrostatic actuators with muscle-like performance,” *Science (80-.)*, vol. 359, no. 6371, pp. 61–65, 2018, doi: 10.1126/science.aao6139.
- [22] L. Maffli, S. Rosset, and H. R. Shea, “Zipping dielectric elastomer actuators: characterization, design and modeling,” *Smart Mater. Struct.*, vol. 22, no. 10, p.

- 104013, Oct. 2013, doi: 10.1088/0964-1726/22/10/104013.
- [23] M. Duranti, M. Righi, R. Vertechy, and M. Fontana, "A new class of variable capacitance generators based on the dielectric fluid transducer," *Smart Mater. Struct.*, vol. 26, no. 11, p. 115014, Nov. 2017, doi: 10.1088/1361-665X/aa8753.
- [24] N. Kellaris, V. Gopaluni Venkata, G. M. Smith, S. K. Mitchell, and C. Keplinger, "Peano-HASEL actuators: Muscle-mimetic, electrohydraulic transducers that linearly contract on activation," *Sci. Robot.*, vol. 3, no. 14, p. eaar3276, Jan. 2018, doi: 10.1126/scirobotics.aar3276.
- [25] M. Taghavi, T. Helps, and J. Rossiter, "Electro-ribbon actuators and electro-origami robots," vol. 9795, no. December, 2018.
- [26] R. H. Baughman, "Playing nature's game with artificial muscles," *Science (80-.)*, vol. 308, no. 5718, pp. 63–5, Apr. 2005, doi: 10.1126/science.1099010.
- [27] X. Wang, S. K. Mitchell, E. H. Rumley, P. Rothmund, and C. Keplinger, "High-Strain Peano-HASEL Actuators," *Adv. Funct. Mater.*, vol. 1908821, p. 1908821, 2019, doi: 10.1002/adfm.201908821.
- [28] C. Cao, X. Gao, and A. T. Conn, "A Magnetically Coupled Dielectric Elastomer Pump for Soft Robotics," *Adv. Mater. Technol.*, vol. 4, no. 8, p. 1900128, Aug. 2019, doi: 10.1002/admt.201900128.
- [29] G. Moretti, M. Duranti, M. Righi, R. Vertechy, and M. Fontana, "Analysis of dielectric fluid transducers," in *Electroactive Polymer Actuators and Devices (EAPAD) XX*, Mar. 2018, vol. 10594, p. 29, doi: 10.1117/12.2297082.
- [30] S. J. A. Koh, C. Keplinger, T. Li, S. Bauer, and Z. Suo, "Dielectric Elastomer Generators: How Much Energy Can Be Converted?," *IEEE/ASME Trans. Mechatronics*, vol. 16, no. 1, pp. 33–41, Feb. 2011, doi: 10.1109/TMECH.2010.2089635.
- [31] R. Pelrine *et al.*, "Dielectric elastomers: Generator mode fundamentals and applications," *Proc. SPIE 4329, Smart Struct. Mater.*, vol. 42, pp. 148–156, 2001.
- [32] G. Kovacs, L. Düring, S. Michel, and G. Terrasi, "Stacked dielectric elastomer actuator for tensile force transmission," *Sensors Actuators, A Phys.*, vol. 155, no. 2, pp. 299–307, 2009, doi: 10.1016/j.sna.2009.08.027.
- [33] P. Lotz, M. Matysek, and H. F. Schlaak, "Fabrication and application of miniaturized dielectric elastomer stack actuators," *IEEE/ASME Trans. Mechatronics*, vol. 16, no. 1, pp. 58–66, 2011, doi: 10.1109/TMECH.2010.2090164.
- [34] T. Mirfakhrai, J. D. W. Madden, and R. H. Baughman, "Polymer artificial muscles," *Mater. Today*, vol. 10, no. 4, pp. 30–38, Apr. 2007, doi: 10.1016/S1369-7021(07)70048-2.
- [35] Z. Li, Y. Wang, C. C. Foo, H. Godaba, J. Zhu, and C. H. Yap, "The mechanism for large-volume fluid pumping via reversible snap-through of dielectric elastomer," *J. Appl. Phys.*, vol. 122, no. 8, p. 084503, Aug. 2017, doi: 10.1063/1.4985827.
- [36] P. Linnebach, G. Rizzello, and S. Seelecke, "Design and validation of a dielectric elastomer membrane actuator driven pneumatic pump," *Smart Mater. Struct.*, vol. 29, no. 7, 2020, doi: 10.1088/1361-665X/ab8a01.
- [37] Y. Chen, L. Agostini, G. Moretti, M. Fontana, and R. Vertechy, "Dielectric elastomer materials for large-strain actuation and energy harvesting: a comparison between styrenic rubber, natural rubber and acrylic elastomer," *Smart Mater. Struct.*, vol. 28, no. 11, p. 114001, Nov. 2019, doi: 10.1088/1361-665X/ab3b32.
- [38] G. Moretti *et al.*, "Resonant wave energy harvester based on dielectric elastomer

- generator,” *Smart Mater. Struct.*, vol. 27, no. 3, 2018, doi: 10.1088/1361-665X/aaab1e.
- [39] M. Hodgins, A. York, and S. Seelecke, “Experimental comparison of bias elements for out-of-plane DEAP actuator system,” *Smart Mater. Struct.*, vol. 22, no. 9, 2013, doi: 10.1088/0964-1726/22/9/094016.
- [40] S. Shian, J. Huang, S. Zhu, and D. R. Clarke, “Optimizing the Electrical Energy Conversion Cycle of Dielectric Elastomer Generators,” *Adv. Mater.*, vol. 26, no. 38, pp. 6617–6621, 2014, doi: 10.1002/adma.201402291.
- [41] R. Kaltseis *et al.*, “Natural rubber for sustainable high-power electrical energy generation,” *RSC Adv.*, vol. 4, no. 53, p. 27905, 2014, doi: 10.1039/c4ra03090g.
- [42] A. Johnsen and K. Rahbek, “A physical phenomenon and its applications to telegraphy, telephony, etc.,” *J. Inst. Electr. Eng.*, vol. 61, no. 320, pp. 713–725, 1923, doi: 10.1049/jiee-1.1923.0092.
- [43] I. A. Anderson, I. A. Ieropoulos, T. McKay, B. O’Brien, and C. Melhuish, “Power for Robotic Artificial Muscles,” *IEEE/ASME Trans. Mechatronics*, vol. 16, no. 1, pp. 107–111, Feb. 2011, doi: 10.1109/TMECH.2010.2090894.
- [44] “LINQTAPE™ PIT1N-Series,” *Technical Datasheet*.
<https://www.caplinq.com/library/technical-data-sheet/#product=175> (accessed Jan. 29, 2020).
- [45] T. L. Sounart, T. A. Michalske, and K. R. Zavadil, “Frequency-dependent electrostatic actuation in microfluidic MEMS,” *J. Microelectromechanical Syst.*, vol. 14, no. 1, pp. 125–133, Feb. 2005, doi: 10.1109/JMEMS.2004.839006.

Acknowledgments

The authors are grateful to Nitto Denko Corporation for providing double sided adhesive tape used for prototypes manufacturing and to CHEROS S.r.l. for providing the PI films used for prototypes manufacturing.

Funding: The research leading to these results received funding from the Italian Ministry of Education, University and Research (MIUR) under the Program Department of Excellence, awarded by the Department of Industrial Engineering of the University of Trento, Italy.

Author contributions: MF conceived the EBM devices and supervised with RV this research. IDS and GM designed the prototypes and carried out the tests. IDS, GM and GB manufactured the prototypes and post-processed the data. MB contributed design and construction of the test-benches. SD and LF contributed to materials and manufacturing procedures definition and selection. All the authors contributed to draft and revised the manuscript.

Competing interests: The authors declare they have no competing interests.

Data and materials availability: Please, contact MF for complete datasets and other materials.

SUPPLEMENTARY MATERIALS

Hardware and materials

EBM Characterization

Movies details

Fig. S1. Model of the EBM.

Fig. S2. Static response of an EBM prototype, according to the model.

Fig. S3. Model validation against experimental data.

Fig. S4. Fabrication of an EBM array.

Fig. S5. Reference dimensions of a single unit circular EBM.

Fig. S6. Schematic of the experimental setup.

Fig. S7. Force variation during actuation tests.

Fig. S8. Single-unit EBM characterization.

Fig. S9. The effect of scaling the radial dimension of an EBM unit.

Fig. S10. Actuation energy density of EBM units as a function of the applied load.

Fig. S11. Cyclic actuation test.

Fig. S12. 1×3 EBM array characterization.

Fig. S13. 1×6 EBM array characterization.

Fig. S14. Comparison of 1×3 and 2×3 EBM arrays.

Fig. S15. Schematic of the pump.

Fig. S16. Comparison of the actual flow with that estimated from the end-effector stroke.

Fig. S17. Pump characterization.

Fig. S18. Energy harvesting scheme

Fig. S19. Actuation-like behavior in the EBM generator.

Tab. S1. Physical and geometrical properties of the EBM design used for simulation and validation.

Tab. S2. Scaling rules for the EBM.

Tab. S3. Features of the developed EBM prototypes.

Movie S1. EBM dynamic tests

Movie S2. Lifting tests

Movie S3. Pump